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ON THE EXPLICIT EXPRESSIONS FOR THE DETERMINATION OF THE FRICTION FACTOR IN TURBULENT REGIME

ACERCA DE LAS EXPRESIONES EXPLÍCITAS PARA LA DETERMINACIÓN DEL COEFICIENTE DE FRICCIÓN EN RÉGIMEN TURBULENTO

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Abstract

Pressure failure in pipelines is commonly calculated by the Darcy-Weisbach formula. To use this formula, the Darcy's hydraulic friction factor must be known. The best approximation for the Darcy's friction coefficient in turbulent flow regime, is expressed by the Colebrook-White equation. This equation can only be solved by numerical methods, because it is implicit for the friction coefficient. There are other approximate explicit models for the calculation with different relative errors in comparison to the Colebrook-White equation. In this work, a review of the friction factor explicit correlations is made including, for the first time in an article of this type, 48 equations, 10 of them that had not been reported previously and other explicit equations whose precision does not agree with previous reports or had not been included in review articles on this topic. The precision of these equations was determined by means of the maximum relative error for Reynolds numbers and relative roughness, considered as the equation's application range. As a result of this work, the evaluation and selection criteria to use the explicit expressions for the calculation of the friction coefficient in turbulent regime is offered to engineering professionals and students.

Keywords: Hydraulic friction coefficient, turbulent regime, explicit models, relative roughness, explicit correlations.

Resumen

La caída de presión en las tuberías se calcula mediante la fórmula Darcy-Weisbach. Para utilizar esta fórmula, debe conocerse el coeficiente de fricción hidráulica de Darcy. La mejor aproximación al coeficiente de fricción de Darcy, para el flujo en régimen turbulento, viene dada por la ecuación de Colebrook-White. Esta ecuación solo puede resolverse mediante métodos numéricos, pues es implícita para el coeficiente de fricción. Hay varios otros modelos explícitos aproximados para su determinación con diferentes errores relativos en comparación con la ecuación de Colebrook-White. En este estudio, se realizó una revisión de las correlaciones explícitas del factor de fricción, y se incluyen, por primera vez en un artículo de este tipo, 48 ecuaciones, de ellas, 10 que no habían sido reseñadas con anterioridad y otras ecuaciones explícitas con cuya precisión no se concuerda con reportes previos, o no se habían incluido en artículos de revisión sobre la temática; se determinó su precisión a través del error relativo máximo, para los números de Reynolds y rugosidades relativas consideradas como su rango de aplicación. Como resultado del trabajo, se brindan a los profesionales y estudiantes de ingeniería, los criterios de evaluación y selección de las expresiones explícitas para el cálculo del coeficiente de fricción en flujo turbulento.

Palabras clave: Coeficiente de fricción hidráulica, régimen turbulento, modelos explícitos, rugosidades relativas, correlaciones explícitas.

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1 Introduction

The determination of the hydraulic friction factor λ value in the general case, depends on the liquid flow regime and on the pipeline material properties. When the flow regime is laminar or turbulent and the pipeline is smooth, the expressions for its calculation coincide and there are no major difficulties; but, when the regime is turbulent, as it is commonly, and the pipeline is rough, there is a great diversity of expressions for the calculation of this coefficient.

The determination of the hydraulic friction factor value λ for the general case, depends on the liquid's regime flow and on the pipeline's material properties. For the calculation of this coefficient in fluid transport systems, which generally operate in turbulent flow regime, there are many formulas proposed.

To calculate λ coefficient in turbulent regime, different authors have proposed a great number of calculation formulas. Initially it was assumed that this coefficient was constant (for example, according to to Dupuit, λ = 0.0025); afterwards various formulas were obtained based in the experimental data processing (Pérez, 2002).

Since 1945, engineers, academics and engineering students have been using a friction factor diagram for the pipeline flow as the type published by Moody, 1947. This diagram is semi-empirical and the conditions that it describes (completely developed flow, isothermal, uncompressible, dissipative, quasi stationary), are never enough reached in practical real cases (McGovern, 2011). Although there has been considerable progress in the calculation of surface roughness, especially with the aid of computational methods, and in the measurement and comprehension of velocities' distribution in the limit layers, the diagram is still considered as a temporary solution and has the function that Moody stated: "… a simple way to estimate friction factors" (McGovern, 2011).

From the bibliographic analysis made in this work, it is observed that there are two types of approaches for the calculation: the implicit calculations, where the Colebrook-White formula is predominant, and the explicit calculations, where there is a great diversity of expressions and disagreements with respect to their complexity and precision, between the researchers that have addressed the theme.

The Colebrook-White formula (Colebrook, 1939), one of the first and more frequently used expressions to calculate the λ coefficient in turbulent regime,

is given by equation (1), where the ε/D relation represents the pipeline's relative roughness.

$$\frac{1}{\sqrt{\lambda}} = -2\log_{10}\left(\frac{\frac{\varepsilon}{D}}{3,7} + \frac{2,51}{Re\sqrt{\lambda}}\right) \tag{1}$$

The use of this formula is not convenient because the model is implicitly defined in the friction factor and iterations are needed for the calculation. To find the friction factor implicitly stated in the Colebrook-White equation, the use of numerical algorithms is needed, which is not as rapid as approximations. Particularly in complex and supercritical pipelines flow systems, its use becomes difficult, and that is the reason why its use is not recommended in practical engineer's calculations or by students in courses related to fluids mechanics.

Difficulties to determine λ by means of Colebrook-White equation, have forced many researchers over the world (Rohsenow, Hartnett, & Cho, 1998), to make efforts in developing explicit equations that could be used alternatively: ones simpler and compact, easier to memorize but with great deviations; others less compact and complex, more difficult to memorize but with small deviations and, some others which combine simplicity and precision, with very reduced errors in the friction factor compared with the calculated value using the Colebrook-White equation (Diniz and Souza, 2009). The explicit expressions for the calculation of the friction coefficient in turbulent flow, most frequently used and cited in literature, are those of Halland and Swamee-Jain (Eq.12 and 20); meanwhile there is an important and interesting group of these equations (Eq. 2, 4, 5, 6, 7, 26, 29, 46, 47 and 48), that are not analyzed in the articles here reviewed, being many of these proposals evaluated as alternatives to the traditionally used ones.

In summary, it was observed that there is no agreement with respect to the expressions' complexity and precision, there is lack of analysis of some expressions cited in literature and also, there is absence of calculation criteria and analysis of the advantages and disadvantages of their use. This lack of information leads to some degree of confusion between technicians and students when they need to choose the expression to use.

As a result of this study, a critical exhaustive review of 48 explicit correlations for the calculation of the hydraulic friction coefficient in turbulent flow is presented, with the objective of analyzing their complexity and precision, including some less known explicit correlations, to provide engineer professionals and engineering students, a set of evaluation and selection criteria for practical uses.

2 Materials and methods

An exhaustive and critical review was made of 68 research works on the subject of the calculation of the turbulent flow friction factor, published in English, Spanish, German, Russian and Portuguese, in recognized indexed journals from data bases in the field of study. The search strategy used as criterion of inclusion, was that consulted sources had to verse on explicit expressions for the calculation or the valuation of their precision, which led to the identification and analysis of 48 friction factor's correlations that cover a wide range of relative roughness and Reynolds numbers. Works related with mathematical methods for the solution of implicit equations for the friction factor calculation and the equations that are valid only for smooth pipelines or non-Newtonian fluids, were excluded. From the use of documentary analysis tools for primary information, it was assumed as a rule of the logic of the results reported in this work, to present the chronological order of publications with conclusions and analytical-synthetic deductions.

3 Results and discussion

Published articles in indexed journals are the main communication medium for the results of research on the friction factor calculation in turbulent flow, as they represent the 88.1% of the total consulted sources (Table 1). Most of the articles have been published in journals in English language, and they represent the 77.6% of the total of consulted articles. These articles appear in many journals, but 8 of these journals publish the 37.8% of the total number of articles (Table 2).

Table 1. Bibliographic sources consu	lted	
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Consulted sources	Quantity
Journal articles	56
Books	8
Reports	2
Conferences	1
Book chapters	1
Non published work	1

Table 2. Journals with major number of publications in the topic.

Journals	Number of articles
Journal of Hydraulic	3
Engineering	
International Journal of Heat	2
and Mass Transfer	
Journal of Petroleum Science	2
and Engineering	
AIChE Journal	2
Advances in Engineering	2
Software	
Journal of Fluids Engineering	2
Journal of the Hydraulics	2
Division	
Industrial & Engineering	2
Chemistry Fundamentals	

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Table 5. Authors	with	major	number	OI	publ	ications.

Authors	Number of articles
Brkić, D.	4
Churchill, S. W.	2
Jain, A. K.	2
Zigrang, D. J.	2
Sylvester, N. D.	2
Offor, U. H.	2
Alabi, S. B.	2



Fig. 1. Distribution of publications per year.

There are many researchers that have published works on the calculation of friction factor in turbulent flow, but there are 7 authors who publish more frequently and have written two or more articles than other researchers on this topic (Table 3). In the last nine decades, from 1940 to our days, there has been an increasing research interest in this topic, especially in the last ten years as, until 2008 there was a uniform, nearly constant, not increasing tendency (Fig. 1).

Equation number	Author (Reference)	Year	Equation	Application range (or valuation)
1	Moody (1947)	1947	$\lambda = 0.0055 \left[1 + \left(2x10^4 \frac{\varepsilon}{D} + \frac{10^6}{Re} \right)^{1/3} \right]$	$4 \times 10^3 \le \text{Re} \le 10^8$ $0 \le \varepsilon/D \le 10^{-2}$
2	Altshul (1963)	1950	$\frac{1}{\sqrt{\lambda}} = 1.8 \log\left(\frac{Re}{0.1Re\frac{\varepsilon}{D} + 7}\right)$	$4 \times 10^{3} \le \text{Re} \le 1 \times 10^{6}$ $1.6 \times 10^{-4} \le \varepsilon/D \le 2.5 \times 10^{-2}$
3	Altshul (1963)	1950	$\lambda = 0.11 \left(\frac{68}{\text{Re}} + \frac{\varepsilon}{D}\right)^{0.25}$	$4 \times 10^{3} \le \text{Re} \le 1 \times 10^{6}$ $1.6 \times 10^{-4} \le \varepsilon/D \le 2.5 \times 10^{-2}$
4	Agroskin (1954)	1951	$\frac{1}{\sqrt{\lambda}} = -1.8.\log\left[\frac{6.8}{Re} + \left(\frac{\varepsilon}{3.7D}\right)^{1.1}\right].$	$\frac{4 \times 10^{3} \le \text{Re} \le 10^{8}}{10^{-6} \le \varepsilon/D \le 10^{-2}}$
5	Frenkel (1956)	1951	$\frac{1}{\sqrt{\lambda}} = -2.\log\left(\left(\frac{6.81}{Re}\right)^{0.9} + \frac{\varepsilon}{3.7D}\right)$	$4 \times 10^{3} \le \text{Re} \le 10^{8}$ $10^{-6} \le \varepsilon/D \le 10^{-2}$
6	Lobaev (1956)	1956	$\lambda = \frac{1.42}{\left[\log\left(\operatorname{Re}\frac{D}{\varepsilon}\right)^2\right]}$	$1 \times 10^{-4} \le \varepsilon/D \le 1 \times 10^{-2}$
7	Chernikin (1958)	1958	$\frac{1}{\sqrt{\lambda}} = -1.83. \log\left[\frac{8.5}{Re} + \left(\frac{\varepsilon}{3.7D}\right)^{1.093}\right]$	$4 \times 10^3 \le \text{Re} \le 10^8$ $10^{-6} \le \varepsilon/D \le 10^{-2}$
8	Wood (1966)	1966	$\lambda = 0.094 \left(\frac{\varepsilon}{D}\right)^{0.225} + 0.53 \left(\frac{\varepsilon}{D}\right) + 88 \left(\frac{\varepsilon}{D}\right)^{0.44} Re^{-\left[1.62 \left(\frac{\varepsilon}{D}\right)^{0.134}\right]}$	$4 \times 10^{3} \le \text{Re} \le 5 \times 10^{7}$ $10^{-5} \le \varepsilon/D \le 4 \times 10^{-2}$
9	Churchill (1973)	1973	$\frac{1}{\sqrt{\lambda}} = -2\log\left[\frac{\varepsilon}{3.7D} + \left(\frac{7}{Re}\right)^{0.9}\right]$	Not specified
10	Eck (1966)	1963	$\frac{1}{\sqrt{\lambda}} = -2log\left(\frac{\varepsilon}{3.715D} + \frac{15}{Re}\right)$	$4 \times 10^{3} \le \text{Re} \le 10^{8} \\ 5 \times 10^{-6} \le \varepsilon/D \le 10^{-2}$
11	Jain (1976)	1976	$\frac{1}{\sqrt{\lambda}} = -2\log\left[\frac{\varepsilon}{3.715D} + \left(\frac{6.943}{Re}\right)^{0.9}\right]$	$5 \times 10^3 \le \text{Re} \le 10^7$ $4x 10^{-5} \le \varepsilon/D \le 5x 10^{-2}$
12	Swamee and Jain (1976)	1976	$\lambda = \frac{0.25}{\left[\log\left(\frac{\varepsilon/D}{3.7} + \frac{5.74}{Re^{0.9}}\right)\right]^2}$	$5 \times 10^3 \le \text{Re} \le 10^8$ $10^{-6} \le \varepsilon/D \le 5 \times 10^{-2}$
13	Churchill (1977)	1977	$\lambda = 8 \left[\left(\frac{8}{Re} \right)^{12} + \frac{1}{(A+B)^{1.5}} \right]^{\frac{1}{12}}$	For all flow regimes and relative roughness
			$A = \left[2.457 \ln \left(\frac{1}{\left(\frac{7}{Re}\right)^{0.9} + \left(\frac{0.27\varepsilon}{D}\right)} \right) \right]^{16}$ $B = \left(\frac{37530}{Re}\right)^{16}$	

Table 4. Explicit expressions for the calculation of the Friction Coefficient in Turbulent Flow.

Equation number	Author (Reference)	Year	Equation	Application range (or valuation)
14	Chen (1979)	1979	$\frac{1}{\sqrt{\lambda}} = -2\log\left[\frac{\varepsilon}{3.7065D} - \frac{5.0452}{Re}\log\left(\frac{1}{2.8257}\left(\frac{\varepsilon}{D}\right)^{1.1098} + \frac{5.850}{Re^{0.89}}\right)^{1.098}\right]$	$ \begin{pmatrix} 16\\ 81 \end{pmatrix} 4 \times 10^3 \le \text{Re} \le 4 \times 10^8 \\ 10^{-7} \le \varepsilon/D \le 5 \times 10^{-2} $
15	Round (1980)	1980	$\frac{1}{\sqrt{\lambda}} = 1.8 \log \left[\frac{Re}{0.135 Re\left(\frac{\varepsilon}{D}\right) + 6.5} \right]$	$4 \times 10^3 \le \text{Re} \le 10^8$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
16	Schorle and col. (1980)	1980	$\frac{1}{\sqrt{\lambda}} = -2log\left[\frac{\varepsilon}{3.7D} - \frac{5.02}{Re}log\left(\frac{\varepsilon}{3.7D} + \frac{14.5}{Re}\right)\right]$	$4 \times 10^3 \le \text{Re} \le 4 \times 10^8$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
17	Barr and White (1981)	1981	$\frac{1}{\sqrt{\lambda}} = -2\log\left\{\frac{\varepsilon}{3.7D} + \frac{4.518\log\left(\frac{Re}{7}\right)}{Re\left[1 + \frac{Re^{0.52}}{29}\left(\frac{\varepsilon}{D}\right)^{0.7}\right]}\right\}$	Not specified
18	Zigrang and Sylvester (1982)	1982	$\frac{1}{\sqrt{\lambda}} = -2\log\left\{\frac{\varepsilon}{3,7D} - \frac{5,02}{\text{Re}}\log\left[\frac{\varepsilon}{3,7D} - \frac{5,02}{\text{Re}}\log\left(\frac{\varepsilon}{3,7D} + \frac{13}{\text{Re}}\right)\right]\right\}$	$\begin{cases} 4 \times 10^3 \le \text{Re} \le 10^8 \\ 4 \times 10^{-5} \le \varepsilon/D \le 5 \times 10^{-2} \end{cases}$
19	Zigrang and Sylvester (1982)	1982	$\frac{1}{\sqrt{\lambda}} = -2,0\log\left\{\frac{\varepsilon/D}{3,7} - \frac{5,02}{Re}\log\left[\frac{\varepsilon/D}{3,7} - \frac{5,02}{Re}\log\left[\frac{\varepsilon/D}{3,7} - \frac{5,02}{Re}\right]\log\left(\frac{\varepsilon/D}{3,7} + \frac{12}{Re}\right)\right\}$	$\frac{4 \times 10^3 \le \text{Re} \le 10^8}{4 \times 10^{-5} \le \varepsilon/D \le 5 \times 10^{-2}}$
20	Haaland (1983)	1983	$\lambda = -1.8 \log \left(\left(\frac{\varepsilon / D}{3.7} \right)^{1.11} + \frac{6.9}{Re} \right)^{-2}$	$\frac{4 \times 10^{3} \le \text{Re} \le 10^{8}}{10^{-6} \le \varepsilon/D \le 5 \times 10^{-2}}$
21	Serghides (1984)	1984	$\frac{1}{\sqrt{\lambda}} = A - \frac{(B-A)^2}{C-2B+A}$ $A = -2log\left[\left(\frac{\varepsilon/D}{3.7}\right) + \frac{12}{Re}\right]$ $B = -2log\left[\left(\frac{\varepsilon/D}{3.7}\right) + \frac{2.51A}{Re}\right]$ $C = -2log\left[\left(\frac{\varepsilon/D}{3.7}\right) + \frac{2.51B}{Re}\right]$	For all the flow regimes and relative roughness
22	Serghides (1984)	1984	$\lambda = \left[4.781 - \frac{(A_7 - 4.781)^2}{A_8 - 2A_7 + 4.781} \right]^{-2}$ $A_7 = -2\log\left(\frac{\varepsilon}{3.7D} + \frac{12}{Re}\right)$ $A_8 = -2\log\left(\frac{\varepsilon}{3.7D} + \frac{2.51A_7}{Re}\right)$	Not specified

Equation number	Author (Reference)	Year	Equation	Application range (or valuation)
23	Tsal (1989)	1989	$A = 0.11 \left(\frac{68}{Re} + \frac{\varepsilon}{D}\right)^{0.25}$ If $A \ge 0.018$ then $\lambda = A$ If $A < 0.018$ then $\lambda = 0.0028 + 0.85A$	$4 \times 10^{3} \le \text{Re} \le 10^{8}$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
24	Robaina (1992)	1992	$\frac{1}{\sqrt{\lambda}} = -2log\left(0.27\frac{\varepsilon}{D} + \frac{5.62}{Re^{0.9}}\right)$	$4 \times 10^{3} \le \text{Re} \le 4 \times 10^{7}$ $10^{-5} \le \varepsilon/D \le 10^{-2}$
25	Manadilli (1997)	1997	$\frac{1}{\sqrt{\lambda}} = -2log\left(\frac{\varepsilon}{3.7D} + \frac{95}{Re^{0.983}} + \frac{96.82}{Re}\right)$	$5,235 \times 10^3 \le \text{Re} \le 10^8$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
26	Chernikin and Chernikin (2012)	1997	$\lambda = 0.11 \left[\frac{\alpha + \delta + X^{1,4}}{115X + 1} \right]^{0.25}$ $\alpha = \frac{68}{Re}; \delta = \frac{k}{D}; X = (28\alpha)^{10}$	For all flow regimes $1.6 \times 10^{-4} \le \varepsilon/D \le 2.5 \times 10^{-2}$
27	Sousa and col. (1999)	1999	$\frac{1}{\sqrt{\lambda}} = -2\log\left[\frac{\varepsilon}{3.7D} - \frac{5.16}{Re}\log\left(\frac{\varepsilon}{3.7D} + \frac{5.09}{Re^{0.87}}\right)\right]$	$4 \times 10^3 \le \text{Re} \le 10^8$ $10^{-6} \le \varepsilon/D \le 10^{-2}$
28	Romeo and col. (2002)	2002	$\frac{1}{\sqrt{\lambda}} = -2\log\left\{\frac{\varepsilon}{3.7065D} - \frac{5.0272}{Re}\log\left[\frac{\varepsilon}{3.827D} - \frac{4.567}{Re}\log(A)\right]\right\}$ $A = \left(\frac{\varepsilon}{7.7918D}\right)^{0.9924} + \left(\frac{5.3326}{208.815\text{Re}}\right)^{0.9345}$	$3 \times 10^{3} \le \text{Re} \le 1.5 \times 10^{8}$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
29	Dobromyslov (2004)	2004	$\sqrt{\lambda} = 0.5 \frac{\frac{b}{2} + \frac{\left(1.312(2-b)\log\left(\frac{3.7D}{\varepsilon}\right)\right)}{\log(Re) - 1}}{\log\left(\frac{3.7D}{\varepsilon}\right)}$ $b = 1 + \frac{\log(Re)}{\log(Re_{kv})}$ If $b > 2$, $b = 2$ $Re_{kv} = 500.\frac{D}{\varepsilon}$	Not specified
30	Sonnad and Goudar (2006)	2006	$\frac{1}{\sqrt{\lambda}} = 0.8686 ln \left(\frac{0.4587 Re}{G^{(G/G+1)}}\right)$ G = 0.1240× $\frac{\varepsilon}{D}$ ×Re + ln (0.4587Re)	$4 \times 10^{3} \le \text{Re} \le 10^{8}$ $10^{-6} \le \varepsilon/D \le 5 \times 10^{-2}$
31	Rao and Kumar (2007)	2006	$\frac{1}{\sqrt{\lambda}} = 2\log\left[\frac{\left(2\frac{\varepsilon}{D}\right)^{-1}}{\left(\frac{0.444 + 0.135Re}{Re}\right)\beta}\right]$ $\beta = 1 - 0.55e^{-0.33\left[\ln\left(\frac{Re}{6.5}\right)\right]^2}$	$2300 \le \text{Re} \le 10^8$ $10^{-6} \le \varepsilon/D \le 5 \times 10^{-2}$

Equation number	Author (Reference)	Year	Equation	Application range (or valuation)
32	Buzzelli (2008)	2008	$\frac{1}{\sqrt{\lambda}} = B_1 - \left[\frac{B_1 + 2\log\left(\frac{B_2}{Re}\right)}{1 + \frac{2.18}{B_2}}\right]$ $B_1 = \frac{\left[0.774\ln(Re)\right] - 1.41}{\left(1 + 1.32\sqrt{\frac{\varepsilon}{D}}\right)}$ $B_2 = \frac{\varepsilon}{3.7D}Re + 2.51B_1$	$3 \times 10^{3} \le \text{Re} \le 1.5 \times 10^{8}$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
33	Vantankhah and Kouchakzadeh (2008)	2008	$\frac{1}{\sqrt{\lambda}} = 0.8686 \ln \left[\frac{0.4587 Re}{(S-0,31)^{\left(\frac{S}{S+0.9633}\right)}} \right]$ $S = 0.124 Re \left(\frac{\varepsilon}{D}\right) + \ln \left(0.4587 Re\right)$	$4 \times 10^{3} \le \text{Re} \le 10^{8}$ $10^{-6} \le \varepsilon/D \le 5 \times 10^{-2}$
34	Avci and Karagoz (2009)	2009	$\lambda = \frac{6.4}{\left\{ ln(Re) - ln \left[1 + 0.01Re \frac{\varepsilon}{D} \left(1 + 10\sqrt{\frac{\varepsilon}{D}} \right) \right] \right\}^{2.4}}$	$2300 \le \text{Re} \le 10^8 \\ 10^{-6} \le \varepsilon/D \le 5 \times 10^{-2}$
35	Papaevangelou and col. (2010)	2010	$\lambda = \frac{0.2479 - 0.0000947 (7 - logRe)^4}{\left[log \left(\frac{\varepsilon}{3.615D} + \frac{7.366}{Re^{0.9142}} \right) \right]^2}$	$10^4 \le \text{Re} \le 10^7 \\ 10^{-5} \le \varepsilon/D \le 10^{-2}$
36	Chernikin and Talipov (2010)	2010	$\lambda = 0,168 \frac{\left(\frac{\varepsilon}{\overline{D}}\right)^{0.102}}{Re^{0.113}}$	$8x10^{-5} \le \varepsilon/D \le 1x10^{-3}$ $10^4 \le \text{Re} \le 4x10^6$
37	Brkić (2011a)	2011	$\frac{1}{\sqrt{\lambda}} = -2\log\left(10^{-0.4343\beta} + \frac{\varepsilon}{3.71D}\right)$ $\beta = \ln\frac{Re}{1.816\ln\left[\frac{1.1Re}{\ln(1+1.1Re)}\right]}$	$4 \times 10^3 \le \text{Re} \le 10^8$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
38	Brkić (2011a)	2011	$\frac{1}{\sqrt{\lambda}} = -2log\left(\frac{2.18\beta}{Re} + \frac{\varepsilon}{3.71D}\right)$ $\beta = ln \frac{Re}{1.816ln\left[\frac{1.1Re}{ln(1+1.1Re)}\right]}$	$4 \times 10^3 \le \text{Re} \le 10^8$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
39	Fang and col. (2011)	2011	$\lambda = 1.613 \left\{ ln \left[0.234 \left(\frac{\varepsilon}{D} \right)^{1.1007} - \frac{60.525}{Re^{1.1105}} + \frac{56.291}{Re^{1.0712}} \right] \right\}^{-2}$	$3 \times 10^3 \le \text{Re} \le 10^8$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
40	Ghanbari and col. (2011)	2011	$\lambda = \left\{ -1.52 log \left[\left(\frac{\varepsilon}{7.21D} \right)^{1.042} + \left(\frac{2.731}{Re} \right)^{0.9152} \right] \right\}^{-2.169}$	$2.1 \times 10^3 \le \text{Re} \le 10^8$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$

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Equation number	Author (Reference)	Year	Equation	Application range (or valuation)
41	Samadianfard (2012)	2012	$\lambda = \left(\frac{Re^{\frac{\varepsilon}{D}} - 0.6315093}}{Re^{\frac{1}{3}} + \text{Re} \cdot \frac{\varepsilon}{D}}\right) + 0.02275308 \left(\frac{6.929841}{Re} + \frac{\varepsilon}{D}\right)^{\frac{1}{9}} + A$ $A = \left(\frac{10^{\frac{\varepsilon}{D}}}{\frac{\varepsilon}{D} + 4.781616}}{\left(\sqrt{\frac{\varepsilon}{D}} + \frac{9.99701}{Re}\right)}\right)$	$4x10^{3} \le \text{Re} \le 10^{8}$ $10^{-6} \le \varepsilon/D \le 5 \times 10^{-2}$
42	Shaikh and col. (2015)	2015	$\lambda = 0.25 \left[log \left(\frac{2.51}{\alpha Re} + \frac{\varepsilon}{3.7D} \right) \right]^{-2}$ $\alpha = \left[1.14 - 2log \left(\frac{\varepsilon}{D} \right) \right]^{-2}$	$10^4 \le \text{Re} \le 10^8$ $10^{-4} \le \varepsilon/D \le 5 \times 10^{-2}$
43	Brkić (2016)	2016	$\frac{1}{\sqrt{\lambda}} = -2\log\left[\frac{2.51\left(1.14 - 2\log\left(\frac{\varepsilon}{D}\right)\right)}{Re} + \frac{\varepsilon}{3.71D}\right]$	$10^4 < \text{Re} < 10^8$ $10^{-6} < \varepsilon/D < 5 \times 10^{-2}$
44	Offor and Alabi (2016a)	2016	$\frac{1}{\sqrt{\lambda}} = -2\log\left\{\frac{\varepsilon}{3.71D} - \frac{1.975}{Re}\left[ln\left(\left(\frac{\varepsilon}{3.93D}\right)^{1.092} + \left(\frac{7.627}{Re+395.9}\right)\right)\right]\right]$	$4 \times 10^3 \le \text{Re} \le 10^8$ $0 \le \varepsilon/D \le 5 \times 10^{-2}$
45	Beluco and Schettini (2016)	2016	$\lambda = \frac{0.3009}{\left[log \left(\left(\frac{\varepsilon}{3.7315D} \right)^{1.0954} + \left(\frac{5.9802}{Re} \right)^{0.9695} \right) \right]^2}$	$3x10^3 < \text{Re} < 9x10^8$ $0 < \varepsilon/D < 9 \times 10^{-2}$
46	Brkić and Praks (2018)	2018	$\lambda = \frac{64}{Re} \cdot (1 - y_1) + \frac{0.316}{Re^{0.25}} \cdot (y_1 - y_3) + \frac{0.25}{log^2 \left(\frac{\varepsilon}{3.71}\right)} \cdot y_2 I$ $\lambda = \frac{64}{Re} \cdot (1 - y_1) + \left(0.0032 + \frac{0.221}{Re^{0.237}}\right) \cdot (y_1 - y_3) + 0.11 \cdot \varepsilon^{0.25} \cdot y_2 I$ $\stackrel{(a)}{\underset{(c_1)}{}}$	For all flow regimes and relative roughness
47	Brkić and Praks (2018)	2018	$\lambda = \frac{64}{Re} \cdot (1 - y_1) + \frac{0.316}{Re^{0.25}} \cdot (y_1 - y_3) + 0.11 \cdot \varepsilon^{0.25} \cdot y_2 \text{III}$ $\lambda = \frac{64}{Re} \cdot (1 - y_1) + \left(0.0032 + \frac{0.221}{Re^{0.237}}\right) \cdot (y_1 - y_3) + \frac{0.25}{\log_{10}^{-2} \left(\frac{\varepsilon}{3.71}\right)} \cdot y_2 \text{IV}$ $y_1 = 1 - \frac{1048}{\frac{4.489}{10^{20}} \cdot Re^6} \cdot \left(0.148 \cdot Re - \frac{2.306 \cdot Re}{0.003133 \cdot Re + 9.646}\right) + 1050$ $y_2 = 1.012 - \frac{1}{0.02521 \cdot Re \cdot \varepsilon} + 2.202$ $y_3 = 1 - \frac{1}{0.000389 \cdot Re^2 \cdot \varepsilon^2} + 0.0000239 \cdot Re + 1.61$	

Equation number	Author (Reference)	Year	Equation	Application range (or valuation)																					
		$\lambda = \left[-2ln \left(\frac{\varepsilon}{\frac{D}{3.7}} + \frac{8.147}{Re^{0.928}} \right) \right]^{-2.096} \cdot \frac{1 + A + B}{1 + C + D}$																							
			$A = a \cdot \left[ln(Re) \right]^{0.01} + b \cdot \left[ln(Re) \right]^{0.1}$ $+ c \cdot ln(Re) + d \cdot ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^2 + f \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^2 + f \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + g \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot \left[ln\left(\frac{\varepsilon}{2}\right) \right]^3 + e \cdot ln(Re) ln\left(\frac{\varepsilon}{2}\right) + e \cdot ln(Re) ln\left($	<u>E</u>																					
Azizi and col	48 Azizi and col. 201 (2018) 201	$_{0}$ Azizi and col.	Azizi and col. 2018 (2018)	$B = h \cdot (Re)^{-0.4} + i \cdot \left(\frac{\varepsilon}{D}\right)^{0.5} + j \cdot (Re)^{0.3} \left(\frac{\varepsilon}{D}\right)^{1.4}$	$D = 10^{-6} \le \frac{\varepsilon}{D} \le 0.05$																				
40		2018		2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	2010	$C = k \ln(Re) + l \left[\ln(Re) \right]^{0.3} + m \ln\left(\frac{\varepsilon}{D}\right) + n \left[\ln\left(\frac{\varepsilon}{D}\right) \right]^2 + o \left[\ln(Re) \right]^{2.3} \ln\left(\frac{\varepsilon}{D}\right)^2$
				$D = p \cdot \left(\frac{\varepsilon}{D}\right)^{0.095} + q \cdot \left(\frac{\varepsilon}{D}\right)^{1.67} + r \cdot \left(\frac{\varepsilon}{D}\right)^{2.6}$																					
			a = 27,6174133619768 b = -25,6353346801891 c = 0.340879778909867 d = 6.0745613327875E 03																						
			e = 4,47951990420369 E-04 f = -6,38574665338776E-05																						
		g= -4,6057036243026E-03 h= -3,180617554529	g= -4,6057036243026E-03 h= -3,18061755452957																						
			1 = 0.329282387073743 = -2.62873832656825E-03 k = 0.057457596107698 = -0.746573202915635																						
			m = -2,18201121395341E-03 n = 4,3570542661899E-04																						
			o = -3,45044088227631E-06 p = 4,43511785774609E-02 a = -1,97409129131745 E-02 r = 0,179513508976549																						
49	Azizi and col. (2018)	2018	$\lambda = \left[1.805 log \left(\frac{\left(\frac{\varepsilon}{D}\right)^{1.108}}{4.267} + \frac{5.164}{Re^{0.966}} \right) \right]^{-2}$	$4x10^{3} \le \text{Re} \le 10^{8}$ $10^{-6} \le \varepsilon/D \le 5 \times 10^{-2}$																					

Many explicit approximations of the loss of charge coefficient with different precision and complexity levels, have been developed -and keep developing-, for the substitution of the Colebrook-White standard implicit equation.

Table 4 shows in chronological order, the 48 explicit expressions found in this study with their application range, including 10 formulas that had not been previously described in none of the review articles consulted. Recent expressions as the proposals for Zeghadnia, Robert & Achour (2019) have not been incorporated, because they consist fundamentally in corrections of some of those already referred to in table 4.

Explicit approximations give a relatively good prediction of the friction factor λ and they can reproduce with precision the Colebrook-White equation and Moody's diagram. Generally, models with more complex approximations are the most accurate (Brkić y Ćojbašić, 2017).

The existing explicit models precisions have been evaluated by three statistical criteria: the mean square error, the percent relative error and the absolute error (Zigrang and Sylvester, 1985; Yildirim, 2009; Jaric *et al.*, 2011; Winning and Coole, 2013), and additionally, by the model of selection criteria (MSC) and the Akaike's information criterion (AIC). The two last criteria were used by Romeo, Royo and Monzon (Romeo *et al.*, 2002) for the selection of the model.

Author (s)	Description	Results
Yildirim (2009)	An extense comparison test was stablished for a wide range of relative roughness (ε/D) and Reynolds (Re) ($1.10^{-6} \le \varepsilon/D \le 5.10^{-2}$; $4.103 \le \text{Re} \le 1.10^8$), that covers a great part of turbulent flow zone in the Moody Diagram.	The majority of the approximations provide estimations of the friction factor with an absolute mean error of $5x10-4$, absolute maximum error of $4x10^{-3}$, mean relative error of 1.3 % and a maximum relative error of 5.8 %; in the complete range of values of ε/D and Re. The comparative analysis for the mean relative error profile, the classification of the six best adjusted equations revised, were in good agreement with those of the best model selection criterion, vindicated in recent literature, for the simulations made.
Jaric <i>et al</i> . (2011)	Review of the most common explicit correlations for the estimation of the friction factor in smooth a rough pipeline. The comparison of friction factor equations with Colebrook-White equation, was expressed by means of the mean relative error, the positive maximum error, the negative maximum error, the negative maximum error, the correlation relation and the standard deviation For the statistical comparison of different equations, the Model of Selection Criterion and the Akaike Information Criterion were used.	It was found that the Zigrang and Sylvester equation gives the most precise value of friction factor and that the Haaland equation is more adequate for manual calculations.
Brkić (2011b)	This article shows a review of the existing explicit approximations comparisons with the Colebrook- White equation.	The majority of the available approximations of the Colebrook-White equation, are very precise. The exceptions are the approximations made by Round, Eck, Moody, Wood, Rao and Kumar. The mean error of nearly all the explicit approximations of Colebrook-White relation is up to 3%.
Salmasi <i>et al.</i> (2012)	The performance of the explicit formulations for the friction factor and the techniques of artificial intelligence (AI) are studied. The AI techniques used include artificial neuronal nets (ANN) and genetic programming (GP). Tests included Re and ε/D transformations, using a logarithmic scale.	This study shows that some of the explicit formulations for the friction factor induce to improper errors, but some of them have god precision The ANN formulation for the resolution of the friction factor is less successful than the GP formulation.

Table 5. Main research work on the topic in the last ten years.

Winning and Coole (2013)	28 explicit equations for the friction factor are revised, their precision with respect to implicit Colebrook-White equation and their relative calculation efficiency are analyzed. For the revision, ranges of Reynolds numbers 4.103 $\leq \text{Re} \leq 4.10^8$ and pipeline relative roughness $10^{-6} \leq \varepsilon/D \leq 10^{-1}$, were selected.	2D and 3D contour graphics were generated which show the range and magnitude of the relative precision percent in the whole range of points for each explicit equation.
Dobrnjac (2012)	Precision and complexity of 15 explicit approximations to the Colebrook-White equation for the determination of the friction factor were studied. The maximum relative error for each approximation was determined and the complexity was valued.	It is demonstrated that the approximations obtained by adjusting Moody Diagram made with C-W formula and Nickuradse measurements, are not successful in the transition flow zone and cover only the turbulent flow above Re=4000. Studies that have eliminated these problems and determine a function of commutation formula for the friction coefficient for all Reynolds numbers Re ($0 \le \text{Re} \le 10^8$) and all relative roughness values, are described. This formula is more precise than C-W formula and then all the other previously published equations
Mohsenabadi <i>et al.</i> (2014)	Report of 29 explicit relations from different researchers which are proved in precision.	It is confirmed that the relations of Serghides (1984), Vatankhah and Kochakzadeh (2008) and Buzzelli (2008), with a relative error below ± 0.05 % for different Reynolds numbers and relative roughness, give the best results. Goudar and Sonnad (2008) and Avci and Karagoz (2009), show the highest level of error in comparison to C-W relation.
Asker et al. (2014)	Various correlations of friction factors are revised. The relative error is evaluated for different Re values and relative pipeline roughness. Statistical analysis is done for each correlation.	It was found that in some of this correlations the percent error is so small that they can be used directly in place of the Colebrook- White equation.
Anaya <i>et al.</i> (2014)	Various mathematical models that describe explicitly the friction factor with respect to Colebrook-White equation and the Kármán number were evaluated. The evaluation was made for a relative roughness value of (ε/D =0.001) and Reynolds numbers between 4.103 and 10 ⁸ .	The use of Pavlov (Frenkel) correlation is recommended.

Lipovka and Lipovka (2014)	A comparative analysis was made of existing formulas for the Darcy's friction factor in turbulent regime and Colebrook-White resolved by the Clamon ¹ method. The absolute mean square deviation is shown in graphics	It is concluded that the Clamond method gives the major precision for all the ranges of ε/D . The second place is occupied by the Goudar and Sonnad equation.
Offor and Alabi (2016b)	The performances of explicit models not based in AI (artificial intelligence) were compared with performances of models based in AI.	The genetic algorithm optimizes the model's explicit parameters, with improvements of 0.12 % to 0.0026 %, according to the index of maximum relative error. Although the genetic programming produces explicit analytical formulas for the determination of output parameters, it was found that they are extremely inaccurate with errors up to 7 % in some models.(Lipovka & Lipovka, 2014)
Lukman and Oke (2018)	Precision of explicit friction factor formulas were evaluated using the relative error, the selection criterion model (SCM) and the Akaike information model (AIM) and were compared with the Colebrook-White formula for Re between entre 4.10^{-3} y 1,704.10 ⁸ y ε/D entre 10^{-7} y 0.052, using Microsoft Excel -Solver.	Formulas with lowest relative error are: Shaikh <i>et al.</i> (0.03%); Serghides (0.70%); Buzzelli (0.70%); Vatankhah and Kouchakzadeh (0.71%); Romeo <i>et al.</i> (0.71%); Sounnad and Goudar (0.71%); Swamee and Jain (0.73%); Barr and White (0.73%); Manadilli (0.73%); Churchill (0.74%); Fang <i>et al.</i> (0.75%); Chen (0.76%); Barr and White (0.77%); Evangelids <i>et al.</i> (0.80%); Zigrang and Sylvester (0.84%); Eck (0.86%); Brikic (0.93%); Jain (0.86%); Haaland (1.52%); Wood (3.48%); Ghanbari <i>et al.</i> (2.17%).
Pimenta <i>et al.</i> (2018)	29 explicit equations found in literature were analyzed and λ was determined by means of Re in the range of 4.103 \leq Re \leq 10 ⁸ and relative roughness (ε/D) of 10 ⁻⁶ $\leq \varepsilon/D \leq 5.10^{-2}$. The performance index and the relative error of formulations were analyzed in relation to the Colebrook-White equation.	The analysis found 7 equations with an excellent performance and high precision, standing out the Offor and Alabi formulation, which can be used as alternative to the standard Colebrook-White equation.

¹The Clamond method is a special algorithm of iterative calculation of λ , which gives accuracy close to limits of computer type double after two iterations. It requires calculation of logarithm once for initial estimation and one time per iteration. (Lipovka and Lipovka, 2014)

The Akaike's information criterion (AIC) is a measure of the relative quality of a statistical model for a set of data. The AIC provides a medium for the model selection. AIC manages a balance between the model's adjustment goodness and the model's complexity. According to this criterion, the most appropriate model is the one that has the smaller value of the AIC. Meanwhile, the model of selection criterion (MSC), is derived from the Akaike's

information criterion and allows a direct comparison between models with a different number of parameters (PN). For this criterion, the most appropriate model is the MSC, bigger, as it propitiates maximizing the model's content of information (Romeo, Royo & Monzón, 2002); while the MSE provides a good indication of the global maximum percent precision of the explicit equation. The peaks in the data are only visualized with the error.

Eq. [#]	Author	Relative	Relative	Computational
		mean error [%]	máximum error [%]	efficiency (Winning y Coole, 2013)
1	Moody (1947)	4.76-7.52	21.5	2
2	Altshul (1963)		16.5*	
3	Altshul (1963)	11.45-16.42	21.5*	1
4	Agroskin (1954)		2.3*	
5	Frenkel (1956)		2.8*	
6	Lobaev (1956)		31.0*	
7	Chernikin (1958)		4.75*	
8	Wood (1966)	3.65-3.88	23.8	3
9	Churchill (1973)	0.08-0.48	2.2	7
10	Eck (1966)	1.50-2.23	8.2	4
11	Jain (1976)	0.18-0.86	2	10
12	Swamee and Jain (1976)	0.04-0.93	2	8
13	Churchill (1977)	0.74	2.2	11
14	Chen (1979)	0.07	0.4	19
15	Round (1980)	3.71-4.47	10.9	6
16	Schorle <i>et al.</i> (1980)	0.13-0.32	0.8	13
17	Barr and White (1981)	0.06	0.3	21
18	Zigrang and Sylvester (1982)	0.00061-0.84	1	15
19	Zigrang and Sylvester (1982)	0.02	0.1	23
20	Haaland (1983)	0.21	1.4	14
21	Serghides (1984)	0.06	0.1	20
22	Serghides (1984)	0.04	0.4	17
23	Tsal (1989)	8.89-16.16	37.5*	5
24	Robaina (1992)	0.84	2.4*	
25	Manadilli (1997)	0.03-0.74	2.1	18
27	Sousa et al. (1999)		0.1*	
28	Romeo et al. (2002)	0.06	0.1	27
29	Dobromyslov (2004)		10.7	
30	Sonnad and Goudar (2006)	0.17	0.8	9
31	Rao and Kumar (2007) 13.27-		82	24
32	Buzzelli (2008) 0.005		0.1	25
33	Vantankhah and Kouchakzadeh (2008)	0.36-0.71	7.6*	8
34	Avci and Karagoz (2009)	1.18-1.72	4.7	12
35	Papaevangelou <i>et al.</i> (2010)	0.23	0.9	28
36	Chernikin and Talipov (2010)		46.0*	
37	Brkić (2011a)	0.12-0.72	2.3	26
38	Brkić (2011a)	0.12-0.48	2.3	22
39	Fang <i>et al.</i> (2011)	0.06	0.4	16
40	Ghanbari et al. (2011)	2.17	3.0*	
41	Samadianfard (2012)	0.08	7	
42	Shaikh <i>et al.</i> (2015)	0.03	10.2	
43	Brkić (2016) 0.49-3.87		8.9*	
44	Offor and Alabi (2016a)	Offor and Alabi (2016a) 0.0025		

Table 6. Precision of the analyzed formulas as reported by previous research.

Eq. [#]	Author	Relative mean error [%]	Relative máximum error [%]	Computational efficiency (Winning y Coole, 2013)
45	Beluco and Schettini (2016)		4.3*	
48	Azizi et al. (2018)		1.6*	

Source: (11, 23, 29, 34, 47, 65, Authors) *For the ranges of Reynolds numbers and relative roughness declared in the table 4.

Observation: Expressions 26, 46 and 47 are not included for being valid for all flow regimes.

Some of the explicit equations have very high errors in a small range of input values, generating peaks that, although being obvious in graphics, could not be so easily identified in data tables and, certainly, cannot be recognized if MSE and maximum relative percent error, have not been reported (Winning and Coole, 2013). These criteria were used by Genić *et al.* (Jaric *et al.*, 2011) and Yildirim (2009) in the comparison of various explicit models.

Brkić (2011a) based on the maximum relative percent error criterion, classified existing explicit models as: extremely precise: with error $\leq 0.14\%$, very precise: error up to 0.5%, moderately precise: error up to 1.5%, less precise: error up to 5%, not recommended: error up to 25% and extremely inexact: error $\geq 80\%$.

The first review on the precision of λ explicit formulations was done in April of 1985 by Gregory and Fogarasi (1985). A second review was made in June of the same year by Zigrang and Sylvester (1985), using the same boundary conditions as used by Serghides (1984). This study was very similar to the one made by Gregory and Fogarasy in relation to the explicit equations revised, with the exception of the rejection of the Wood equation (Wood,1966) and the inclusion of Chen equation (Chen,1979). Goudar and Sonnad (2007, 2008) made other two revisions.

Table 5 shows the main research works done in the last 10 years, on precision and complexity of the different explicit expressions. It is necessary to point out that, all the work of review on the precision of explicit equations for the calculation of the friction factor, has been done for extremely broad ranges of Reynolds numbers and relative roughness, which are not found in common practice.

Yildirim (2009) made another study based in the same matrix of values used by Goudar and Sonnad. Brkić (2011b) made a revision of 20 equations and in the same year another revision of 16 equations in the same range of values as the ones of Goudar and Sonnad and Yildirim, was made. These studies were made with a very large matrix of a million points, which introduced the Altshul equation, cited by Jaric et al. (2011). This revision, as the work of Romeo et al. (2002) used the Selection Criteria Model (SCM) and the Akaike Information Criteria (AIC), to make a statistical comparison of the relative computational efficiency that concluded in the recommendation of using Zigrang and Sylvester (1982) equation. Beluco and Schettini (2016) analyzed six classic approximations of Colebrook-White equation and they also propose a generic model for this category, which results equally complex for reiterative calculations in the solution of fluid mechanics problems. Jaric et al. (2011) stated that the Zigrang and Sylvester equation (Zigrang and Sylvester, 1982) gives the most accurate value of the friction factor and that the Haaland (1983) equation, with similar complexity as the Swamee and Jain equation (1976), is the most appropriate for manual calculations. Pimenta et al. (2018) conclude that the equations (Chen, 1979) and those of Sonnad and Goudar (2006), Buzzelli (2008), Vantankhah and Kouchakzadeh (2008), Fang et al. (2011) and Offor and Alabi (2016b), have high precision in comparison to the Colebrook-White approximation.

Table 6 shows the mean and maximum relative error of the expressions reported by different authors cited in previous reviews, in respect to Colebrook-White equation.

Table 6 also shows the calculated errors for the formulas reported for the first time by this review and other recalculated values in which there is no agreement with previous reports or had not been reported (equations 2, 3, 4, 5, 6, 7, 23, 24, 27, 33, 36, 40, 44 and 45).

From the analysis made, which is shown in Table 6, three tendencies in the development of these expressions can be observed: the first one, is the use of algebraic equations (1, 3, 8, 26, 36, 41); the second one, is the use of expressions that include base 10 logarithm -the most abundant- (equations: 2, 4, 5, 6, 7, 9, 10, 11, 12, 14, 15, 16, 17, 18, 19, 20, 21, 22, 23, 24, 25, 27, 28, 29, 31, 35, 40, 42, 43, 45, 48); and the third one, those which use the natural logarithm (equations: 13, 30, 32, 33, 34, 37, 38, 39, 44, 46, 47).

Order number	Equation number	Author (Reference)	Equation
1	27	Sousa and col. (1999)	$\frac{1}{\sqrt{\lambda}} = -2log\left[\frac{\varepsilon}{3.7D} - \frac{5.16}{Re}log\left(\frac{\varepsilon}{3.7D} + \frac{5.09}{Re^{0.87}}\right)\right]$
2	22	Serghides (1984)	$\lambda = \left[4.781 - \frac{\left(A_7 - 4.781\right)^2}{A_8 - 2A_7 + 4.781} \right]^{-2}$ $A_7 = -2\log\left(\frac{\varepsilon}{12} + \frac{12}{12}\right)$
			$A_{8} = -2\log\left(\frac{\varepsilon}{3.7D} + \frac{2.51A_{7}}{Re}\right)$
3	30	Sonnad and Goudar (2006)	$\frac{1}{\sqrt{\lambda}} = 0.8686 ln \left(\frac{0.4587 Re}{G^{(G/G+1)}} \right)$
			$G = 0.1240 \times \frac{\varepsilon}{D} \times Re + \ln(0.4587Re)$
4	20	Haaland (1983)	$\lambda = -1.8 \log\left(\left(\frac{\varepsilon/D}{3.7}\right)^{1.11} + \frac{6.9}{Re}\right)^{-2}$
5	48	Azizi and col. (2018)	$\lambda = \left[1.805 log \left(\frac{\left(\frac{\varepsilon}{D}\right)^{1.108}}{4.267} + \frac{5.164}{Re^{0.966}} \right) \right]^{-2}$
6	9	Churchill (1973)	$\frac{1}{\sqrt{\lambda}} = -2\log\left[\frac{\varepsilon}{3.7D} + \left(\frac{7}{Re}\right)^{0.9}\right]$
7	4	Agroskin (1954)	$\frac{1}{\sqrt{\lambda}} = -1.8.\log\left[\frac{6.8}{Re} + \left(\frac{\varepsilon}{3.7D}\right)^{1.1}\right].$
8	5	Frenkel (1956)	$\frac{1}{\sqrt{\lambda}} = -2.\log\left(\left(\frac{6.81}{Re}\right)^{0.9} + \frac{\varepsilon}{3.7D}\right)$
9	12*	Swamee and Jain (1976)	$\lambda = \frac{0.25}{\left[log\left(\frac{\varepsilon/D}{3.7} + \frac{5.74}{Re^{0.9}}\right)\right]^2}$
10	34	Avci and Karagoz (2009)	$\lambda = \frac{6.4}{\left\{ ln(Re) - ln \left[1 + 0.01Re \frac{\varepsilon}{D} \left(1 + 10\sqrt{\frac{\varepsilon}{D}} \right) \right] \right\}^{2.4}}$
11	7	Chernikin (1958)	$\frac{1}{\sqrt{\lambda}} = -1.83.\log\left[\frac{8.5}{Re} + \left(\frac{\varepsilon}{3.7D}\right)^{1.093}\right]$

Table 7. Equations with relative errors below 5% for turbulent flow in pipelines with relative roughness $(10^{-6} \le \varepsilon/D \le 5 \times 10^{-2})$ and Reynolds numbers $(10^{-6} \le \varepsilon/D \le 5 \times 10^{-2})$, ordered according to their precision.

* The Swamme-Jain equation shows a maximum relative error of 3.12% in this range, for the rest of the equations, it is shown in Table 6.

Table 8. Valid equations (relative errors less than 5%) for turbulent flow in smooth pipelines or pipelines with low relative roughness ($0 \le \varepsilon/D \le 5 \times 10^{-2}$) and Reynolds numbers ($4 \times 10^3 \le Re \le 10^8$), ordered according to their precision.

Order number	Equation number	Author (Reference)	Equation
1 28	28	Romeo and col. (2002)	$\frac{1}{\sqrt{\lambda}} = -2\log\left\{\frac{\varepsilon}{3.7065D} - \frac{5.0272}{Re}\log\left[\frac{\varepsilon}{3.827D} - \frac{4.567}{Re}\log\left(A\right)\right]\right\}$
	20		$A = \left(\frac{\varepsilon}{7.7918D}\right)^{0.9924} + \left(\frac{5.3326}{208.815 \mathrm{Re}}\right)^{0.9345}$
2	32	Buzzelli (2008)	$\frac{1}{\sqrt{\lambda}} = B_1 - \left[\frac{B_1 + 2\log\left(\frac{B_2}{Re}\right)}{1 + \frac{2.18}{B_2}}\right]$ $B_1 = \frac{\left[0.774\ln(Re)\right] - 1.41}{\left(1 + 1.32\sqrt{\frac{\varepsilon}{D}}\right)}$ $B_2 = \frac{\varepsilon}{Re} + 2.51B$
			$\frac{D_2 - 3.7D}{3.7D} = \frac{1000}{3.7}$
3	44	Offor and Alabi (2016a)	$\frac{1}{\sqrt{\lambda}} = -2\log\left\{\frac{\varepsilon}{3.71D} - \frac{1.975}{Re}\right[\ln\left(\left(\frac{\varepsilon}{3.93D}\right)^{1.02} + \left(\frac{7.627}{Re+395.9}\right)\right)\right]\right\}$
4	39	Fang and col. (2011)	$\lambda = 1.613 \left\{ ln \left[0.234 \left(\frac{\varepsilon}{D} \right)^{1.1007} - \frac{60.525}{Re^{1.1105}} + \frac{56.291}{Re^{1.0712}} \right] \right\}^{-2}$
5	16	Schorle and col. (1980)	$\frac{1}{\sqrt{\lambda}} = -2\log\left[\frac{\varepsilon}{3.7D} - \frac{5.02}{Re}\log\left(\frac{\varepsilon}{3.7D} + \frac{14.5}{Re}\right)\right]$
6	25	Manadilli (1997)	$\frac{1}{\sqrt{\lambda}} = -2log\left(\frac{\varepsilon}{3.7D} + \frac{95}{Re^{0.983}} + \frac{96.82}{Re}\right)$
7	37	Brkić (2011a)	$\frac{1}{\sqrt{\lambda}} = -2\log\left(10^{-0.4343\beta} + \frac{\varepsilon}{3.71D}\right)$ $\beta = \ln \frac{Re}{1.816\ln\left[\frac{1.1Re}{\ln(1+1.1Re)}\right]}$
8	38	Brkić (2011a)	$\frac{1}{\sqrt{\lambda}} = -2\log\left(\frac{2.18\beta}{Re} + \frac{\varepsilon}{3.71D}\right)$
			$\beta = ln \frac{Re}{1.816ln \left[\frac{1.1Re}{ln(1+1.1Re)}\right]}$
9	40	Ghanbari and col. (2011)	$\lambda = \left\{ -1.52 log \left[\left(\frac{\varepsilon}{7.21D} \right)^{1.042} + \left(\frac{2.731}{Re} \right)^{0.9152} \right] \right\}^{-2.169}$
10	45	Beluco and Schettini (2016)	$\lambda = \frac{0.3009}{\left[log \left(\left(\frac{\varepsilon}{3.7315D} \right)^{1.0954} + \left(\frac{5.9802}{Re} \right)^{0.9695} \right) \right]^2}$



Fig. 2. Relative error of explicit expressions selected in accordance to the Colebrook equation (épsilon representa el valor de ε/D).

In table 7 appear the 11 equations that have relative errors lower than 5% valid for turbulent flow in rough pipelines $(10^{-6} \le \varepsilon/D \le 5 \times 10^{-2})$ and Reynolds numbers in the range $4 \times 10^3 \le Re \le 10^8$, which have been ordered according to their precision. Equations of Sousa *et al.* (1999), Serghides (1984) and Sonnad and Goudar (2006), have a relative error lower than 1% and are the most precise ones; among these, the equation of Sousa *et al.* (1999), is the less complex one.

Meanwhile, for smooth pipelines or pipelines with low relative roughness ($0 \le \varepsilon/D \le 5 \times 10^{-2}$), there are 10 equations with relative errors lower than 5% which are valid for turbulent flow and Reynolds numbers ($4 \times 10^3 \le Re \le 10^8$) (table 8). The equations of Romeo *et al.* (2002), Buzzelli (2008), Offor and Alabi (2016a), Fang *et al.* (2011) y Schorle *et al.* (1980), have a relative error lower than 1% and are the most precise ones; but nevertheless, they are quite complex, except the Schorle *et al.* (1980) equation, whose complexity is relatively low.

The main conclusion on the precision of explicit expressions for the calculation of hydraulic friction coefficient in turbulent flow is, that the relative error is not uniformly distributed in the Reynold number's domain (Re) and the relative roughness (ε/D) (Brkić and Ćojbašić, 2017). In figure 2 the formulas of Agroskin (Eq. 4), Frenkel (Eq. 5), Altshul simple (Eq. 3) and Altshul logarithmic (Eq. 2), result vey imprecise for high values of relative roughness; but nevertheless, for relative roughness values lower than 1×10^{-2} , the two first formulas have significantly lower error and give acceptable results for engineering calculations.

On the other hand, the behavior of equations of Chernikin (Eq. 7) and Robaina (Eq. 24), is different, because the major values of relative error are observed in pipelines with small values of relative roughness (Fig. 2).

The great majority of the most recognized authors in fluid mechanics, do not agree in the recommendation of an equation for the calculation of λ in turbulent regime. Çengel and Cimbala (2017) state that for the explicit approximate relation given by Haaland (1983), results are in the 2% of the ones obtained by the Colebrook-White equation and less than 1% with respect to the equation of Churchill (1977).

Mott and Untener (2015) consider the Swamee and Jain (1976) equation, as it allows the direct calculation of λ value for turbulent flow and gives values that are in $\pm 1\%$ into the range of relative roughness: $\varepsilon/d = 0.01$ a 1.10^{-6} and for Reynolds

numbers of 5.10^3 a 10^8 . This is virtually all the turbulent zone of the Moody diagram. The authors state that, although the equations of Agroskin (4), Frenkel (5) and Robaina (24), have similar complexity to the equations of Haaland y Swamee-Jain, which are the most frequently used in Western countries for practical calculations; their precision, lower to these last ones, does not merit its use. Equations of Sousa et al. (27) and Offor and Alabi (44), are very precise, but their use is not recommended owed to their complexity. Because of its simplicity, the possible use of the simple algebraic equation of Altshul (Eq. 3), should be evaluated for the common ranges that are used in practical cases, although for wide ranges of Reynolds numbers and relative roughness, it shows a high imprecision.

Conclusions

- 1. In this study the explicit equations proposed by different researchers, to approximate the values obtained by the Colebrook-White implicit equation were identified and compiled, as well as the precision reported in the literature and the relative computational efficiency of more than 45 explicit relations, and it was confirmed that there is discrepancy in the data given by different authors about the precision and complexity of these equations. From these results, it is recommended to use the equations of Haaland and Swamee-Jain, because of their acceptable precision and medium complexity.
- 2. In this research, the relative maximum error of various explicit equations not included in previous reports, was determined and the same error of other equations that had been studied previously, was adjusted. It was confirmed once again that, the higher the precision in comparison to the Colebrook-White equation, higher will be the complexity of the expressions. Likewise, the relative error varies for the different values of relative roughness and Reynolds numbers.
- 3. The work developed on the explicit equations precision has been done for an extremely wide range of Reynolds numbers and relative roughness, which are not of common use in the engineers practice and in the study of Fluid Mechanics, so it is necessary to determine the

precision of explicit equations for a smaller range of Re and relative roughness, and on this base, propose which of these equations could be more adequate to be used in practical cases.

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Nomenclature

Re: Reynolds Number, dimensionless D: Interior pipeline diameter, m

Greek symbols

 λ : Friction factor or Friction Coefficient, dimensionless ε Pipeline absolute roughness, m

References

- Agroskin, I. I. (1954). *Hydraulics* (Russian). Available in: https://books.google.com.ec/ books?id=nPGpnQEACAAJ. Accesado: 25 agosto 2018. 484 p.
- Altshul, A. D. (1963). *Hydraulic Friction Losses in the Piping*. Gosenergoizdat, Moscow, 256 p.
- Anaya Durand, A. I., Cauich Segovia, G. I., Funabazama Bárcenas, O., Alfonso Gracia, M., Bravo, V. (2014). Evaluación de ecuaciones de factor de fricción explícito para tuberías. *Educación Química* 25, 128-134. https://doi.org/10.1016/S0187-893X(14)70535-X.
- Asker, M., Turgut, O. E., Coban, M. T. (2014). A review of non iterative friction factor correlations for the calculation of pressure drop in pipes. *Bitlis Eren Univ J Sci Technol* 4, 1-8. https://doi.org/10.17678/beujst.90203.
- Avci, A., Karagoz, I. (2009).А novel explicit equation for friction factor in and rough pipes. Journal smooth of

Fluids Engineering 131, 061203-061204. https://doi.org/10.1115/1.3129132.

- Azizi, N., Homayoon, R., Hojjati, M. R. (2018). Predicting the Colebrook-White friction factor in the pipe flow by new explicit correlations. *Journal of Fluids Engineering 141*, 051201-051208. https://doi.org/10.1115/1.4041232.
- Barr, D., White, C. (1981). Technical note. solutions of the colebrook-white function for resistance to uniform turbulent flow. *Proceedings of the Institution of Civil Engineers* 71, 529-535. https://doi.org/10.1680/iicep.1981.1895.
- Beluco, B. C., Schettini, E. (2016). An improved expression for a classical type of explicit approximation of the Colebrook White equation with only one internal iteration. *International Journal of Hydraulic Engineering* 5, 19-23. https://doi.org/10.5923/j.ijhe.20160501.03.
- Brkić, D. (2011a). An explicit approximation of Colebrook's equation for fluid flow friction factor. *Petroleum Science and Technology* 29, 1596-1602. https://doi.org/10.1080/1091646100 3620453.
- Brkić, D. (2011b). Review of explicit approximations to the Colebrook relation for flow friction. *Petroleum Science and Technology* 77, 34-48. https://doi.org/10.1016/j.petrol.2011.02.006.
- Brkić, D. (2016). A note on explicit approximations to Colebrook's friction factor in rough pipes under highly turbulent cases. *International Journal of Heat and Mass Transfer 93*, 513-515. https://doi.org/10.1016/J.IJHEATMASSTRANS FER.2015.08.109.
- Brkić, D., Ćojbašić, Ž. (2017). Evolutionary optimization of Colebrook's turbulent flow friction approximations. *Fluids* 2, 1-15. https://doi.org/10.3390/fluids2020015.
- Brkić, D., Praks, P. (2018). Unified friction formulation from laminar to fully rough turbulent flow. https://doi.org/10.3390/fluids2020015.
- Buzzelli, D. (2008). Calculating friction in one step. *Machine Design 80*, 54-55.
- Çengel, Y. A., Cimbala, J. M. (2017). Fluid Mechanics: Fundamentals and Applications. McGraw-Hill Education, New York, 1024 p.

- Chen, N. H. (1979). An explicit equation for friction factor in pipe. *Industrial & Engineering Chemistry Fundamentals 18*, 296-297. https://doi.org/10.1021/i160071a019.
- Chernikin, V. I. (1958). *Pumping Viscous and Freezing Oils* (Russian). Available in: https://books.google.com.ec/books?id=DnjXPg AACAAJ. Accesado: 15 octubre 2018.
- Chernikin, V., Chernikin, A. (2012). Generalized formula for the calculation of the coefficient of hydraulic resistance of main pipelines for light petroleum products and low-viscosity oil. (Russian). Science and Technologies of Pipeline Transport of Oil and Petroleum Products 4, 64-66.
- Chernikin, A., Talipov, R. F. (2010). On the use of the Colebrook equation for the hydraulic calculation of pipes according to the generalized formula (Russian). *Transport by Pipes 4*, 14-6.
- Churchill, S. W. (1973). Empirical expressions for the shear stress in turbulent flow in commercial pipe. *AIChE Journal 19*, 375-376. https://doi.org/10.1002/aic.690190228.
- Churchill, S. W. (1977). Friction-factor equation spans all fluid-flow regimes. *Chemical Engineering Journal 84*, 91-92.
- Colebrook, C. F. (1939). Turbulent flow in pipes, with particular reference to the transition region between the smooth and rough pipe laws. *Journal of the Institution of Civil Engineers 11*, 133-156. https://doi.org/10.1680/ijoti.1939.13150.
- Diniz, V., Souza, P. A. (2009). Four explicit formulae for friction factor calculations in pipe flow. *WIT Transactions on Ecology and the Environment 125*, 369-380. https://doi.org/10.2495/WRM090331.
- Dobrnjac, M. (2012). Determination of friction coefficient in transition flow region for waterworks and pipelines calculation. *Annals of Faculty Engineering Hunedoara - International Journal of Engineering 10*, 137-142.
- Dobromyslov, A. Y. (2004). Tables for hydraulic calculations of pressure pipes made of polymeric materials (Russian). *VNIIMP*, Moscow, 209 p.

- Eck Pecornik, B. (1973). *Technische Stromungslehre*. New York: Springer, 324 p. https://doi.org/10.10 07/978-3-662-13104-6_1.
- Fang, X., Xu, Y., Zhou, Z. (2011). New correlations of single-phase friction factor for turbulent pipe flow and evaluation of existing singlephase friction factor correlations. *Nuclear Engineering and Design 241*, 897-902. https://doi.org/10.1016/j.nucengdes.2010.12.019.
- Frenkel, N. Z. (1956). *Hydraulics* (Russian). Gosenergizdat, Moscow, 456 p.
- Ghanbari, A., Fred, F., Rieke, H. (2011). Newly developed friction factor correlation for pipe flow and flow assurance. *Journal of Chemical Engineering and Materials Science* 2, 83-86. https://doi.org/10.1109/ICTTA.2004.1307856.
- Goudar, C. T., Sonnad, J. R. (2007). Explicit friction factor correlation for turbulent flow in rough pipe. *Hydrocarbon Process* 86, 103-105. https://doi.org/10.1021/ie0300676.
- Goudar, C. T., Sonnad, J. R. (2008). Comparison of the iterative approximations of the Colebrook-White equation.*Hydrocarbon Processing* 87, 79-83.
- Gregory, G. A., Fogarasi, M. (1985). Alternate to standard friction factor equation. *Oil Gas Journal 83*, 120-127.
- Haaland, S. E. (1983). Simple and explicit formulas for the friction factor in turbulent pipe flow. *Journal of Fluids Engineering 105*, 89. https://doi.org/10.1115/1.3240948.
- Jain, A. K. (1976). Accurate Explicit Equation for Friction Factor. Journal of the Hydraulics Division, 102(5), 674-677.
- Jaric, M., Kolendić, P., Jarić, M., Budimir, N., Genić, V. (2011). A review of explicit approximations of Colebrook's equation. *Faculty of Mechanical Engineering 39*, 67-71.
- Lipovka, A. Y., Lipovka, Y. L. (2014). Determining hydraulic friction factor for pipeline systems. Journal of Siberian Federal University Engineering & Technologies 1, 62.
- Lobaev, B. N. (1956). Calculation of Water and Steam Heating System Piping (Russian). Gosstroyizdat, Kiev, 122 p.

- Lukman, S., Oke, I. (2017). Accurate solutions of Colebrook-White's friction factor formulae. *Nigerian Journal of Technology 36*, 1039-1048. https://doi.org/10.4314/njt.v36i4.8.
- Manadilli, G. (1997). Replace implicit equations with signomial functions. *Chemical Engineering Journal 104*, 129-130.
- McGovern, J. (2011). *Friction Factor Diagrams for Pipe Flow*. Available in: https://arrow.dit.ie/engs chmecart/28. Access: 12 agosto 2018.
- Mohsenabadi, S. K., Biglari, M.R., Moharrampour, M. (2014). Comparison of explicit relations of darcy friction measurement with Colebrook-White equation. *Applied mathematics in Engineering, Management and Technology 2*, 570-578.
- Moody, L. F. (1947). An approximate formula for pipe friction factors. *Transactions of the ASME* 69, 1005-1006.
- Mott, R. L., Untener, J. A. (2015). *Mecánica de Fluidos* (7a. ed.). Pearson Educación, México, 644 p.
- Offor, U. H., Alabi, S. B. (2016a). An accurate and computationally efficient explicit friction factor model. *Advances in Chemical Engineering and Science* 06, 237-245. https://doi.org/10.4236/aces.2016.63024.
- Offor, U. H., Alabi, S. B. (2016b). Performance evaluation of the explicit approximations of the implict colebrook equation. *International Journal of Research in Engineering and Technology 5*, 1-12.
- Papaevangelou, G., Evangelides, C., Tzimopoulos, C. (2010). A new explicit equation for the friction coefficient in the Darcy-Weisbach equation. Presentación PRE10, 6-9 julio, Corfu, Greece. Proceedings of the Tenth Conference on Protection and Restoration of the Environment 166, 1-7 p.
- Pérez Franco, D. (2002). Evolución histórica de las fórmulas para expresar las pérdidas de carga en tuberías. Segunda parte: Desde los trabajos de Darcy hasta los de Stanton. *Ingeniería Hidráulica y Ambiente XXIII*, 3-8.

- Pimenta, B. D., Robaina, A. D., Peiter, M. X., Mezzomo, W., Kirchner, J.H., Ben L., H. B. (2018). Performance of explicit approximations of the coefficient of head loss for pressurized conduits. *Revista Brasileira de Engenharia Agrícola Ambiental Campina Grande* 22, 301-307. https://doi.org/10.1590/1807-1929/agriambi.v22n5p301-307.
- Rao, A.R., Kumar, B. (2007). Friction factor for turbulent pipe flow. Division of Mechanical Science, Civil Engineering. Indian Institute of Science, Bangalore, India. Avalaible in: http://eprints.iisc.ernet.in/9587/
- Robaina, A. D. (1992). Análise de equações explicitas para o cálculo do coeficiente "f" da fórmula universal de perda de carga. *Ciência Rural* 22, 157-159. https://doi.org/10.1590/S0103-8478199200020 0006.
- Rohsenow, W. M., Hartnett, J. P. (James P. ., & Cho, Y. I. (1998). *Handbook of Heat Transfer*. McGraw-Hill.
- Romeo, E., Royo, C., Monzón, A. (2002). Improved explicit equations for estimation of the friction factor in rough and smooth pipes. *Chemical Engineering Journal* 86, 369-374. https://doi.org/10.1016/S1385-8947(01)00254-6.
- Round, G. F. (1980). An explicit approximation for the friction factor-reynolds number relation for rough and smooth pipes. *The Canadian Journal of Chemical Engineering* 58, 122-123. https://doi.org/10.1002/cjce.5450580119.
- Salmasi, F., Khatibi, R., Ghorbani, M. A. (2012). A study of friction factor formulation in pipes using artificial intelligence techniques and explicit equations. *Turkish Journal of Engineering and Environmental Sciences 36*, 121-138. https://doi.org/10.3906/muh-1008-30.
- Samadianfard, S. (2012). Gene expression programming analysis of implicit Colebrook-White equation in turbulent flow friction factor calculation. *Journal of Petroleum Science and Engineering* 92-93, 48-55. https://doi.org/10.1016/j.petrol.2012.06.005.
- Schorle, B. J, Churchill, S. W., Shacham, M. (1980). An explicit equation for

friction factor in pipe. *Industrial & Engineering Chemistry Fundamentals 19*, 228. https://doi.org/10.1021/i160074a019.

- Serghides, T. K. (1984). Estimate friction factor accurately. *Chemical Engineering* 91, 63-64.
- Shaikh, M. M., Massan, S.R., Wagan, A. I. (2015). A new explicit approximation to Colebrook's friction factor in rough pipes under highly turbulent cases. *International Journal of Heat and Mass Transfer 88*, 538-543. https://doi.org/10.1016/J.IJHEATMASSTRANS FER.2015.05.006.
- Sonnad, J.R., Goudar, C. T. (2006). Turbulent Flow Friction Factor Calculation Using a Mathematically Exact Alternative to the Colebrook-White Equation. *Journal* of Hydraulic Engineering 132, 863-867. https://doi.org/10.1061/(ASCE)0733-9429(2006)132:8(863).
- Sousa, J., Da Conceição, M., Marques, A. S. (1999). An explicit solution of the Colebrook-White equation through simulated annealing. *Water Industry Systems: Modelling and Optimization Applications 2*, 347-355.
- Swamee, P.K., Jain, A. K. (1976). Explicit equations for pipe flow problems. *Journal of the Hydraulics Division 102*, 657-664.
- Tsal, R. J. (1989). Altshul-Tsal friction factor equation. *Heating, Piping and Air Conditioning* 8, 30-45.
- Vatankhah, A.R., Kouchakzadeh, S. (2008). Discussion of "Turbulent Flow Friction Factor

Calculation Using a Mathematically Exact Alternative to the Colebrook-White Equation" by Jagadeesh R. Sonnad and Chetan T. Goudar. *Journal of Hydraulic Engineering 134*, 1187. https://doi.org/10.1061/(ASCE)0733-9429(2008)134:8(1187).

- Winning, H. K., Coole, T. (2013). Explicit friction factor accuracy and computational efficiency for turbulent flow in pipes. *Flow, Turbulent Combustion 90*, 1-27. https://doi.org/10.1007/s10494-012-9419-7.
- Wood, D. J. (1966). An explicit friction factor relationship. *Civil Engineering* 36, 60-61.
- Yildirim, G. (2009). Computer-based analysis of explicit approximations to the implicit Colebrook-White equation in turbulent flow friction factor calculation. Advances in Engineering Software 40, 1183-1190. https://doi.org/10.1016/j.advengsoft.2009.04.004.
- Zeghadnia, L., Robert, J. L., & Achour, B. (2019). Explicit solutions for turbulent flow friction factor: A review, assessment and approaches classification. *Ain Shams Engineering Journal*. https://doi.org/10.1016/J.ASEJ.2018.10.007.
- Zigrang, D. J., Sylvester, N. D. (1982). Explicit approximations to the solution of Colebrook's friction factor equation. *AIChE Journal 28*, 514-5. https://doi.org/10.1002/aic.690280323.
- Zigrang, D. J., Sylvester, N. D. (1985). A review of explicit friction factor equations. *Journal* of Energy Resources Technology 107, 280. https://doi.org/10.1115/1.3231190.